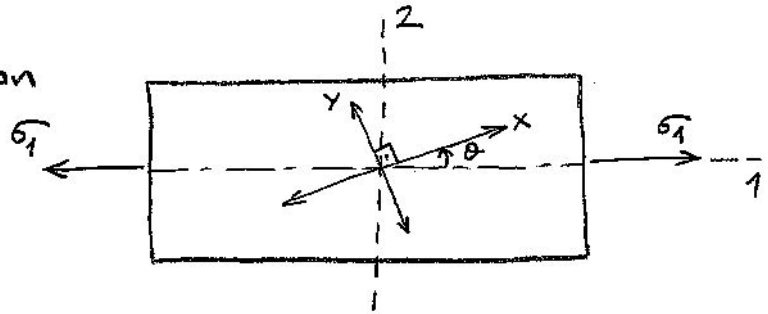
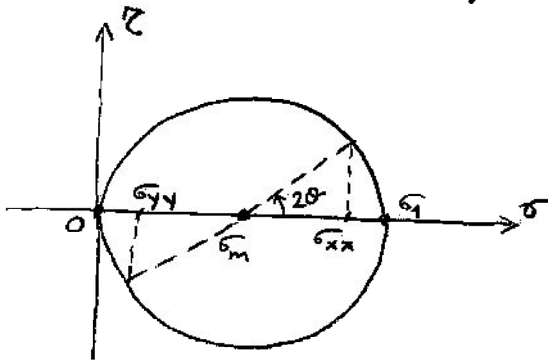


Problem 1

In a uniaxial tension/compression experiment we are measuring  $\epsilon_{xx}$  and  $\epsilon_{yy}$  as shown in the sketch ( $\theta \rightarrow$  unknown).



We can quickly analyze this situation using Mohr's circle for uniaxial loading



mean stress:  $\sigma_m = \frac{\sigma_{xx} + \sigma_{yy}}{2}$

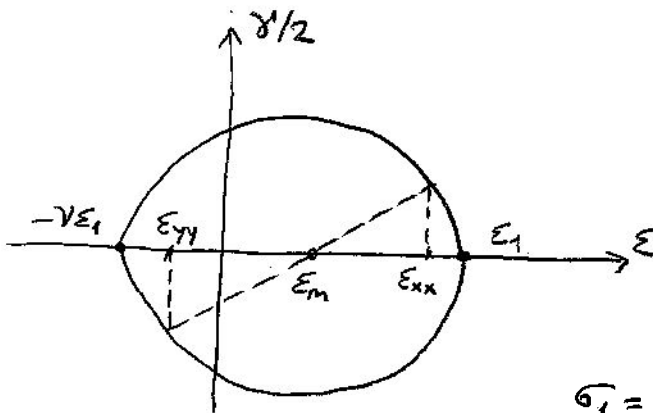
radius:  $R = \sigma_m = \frac{\sigma_{xx} + \sigma_{yy}}{2}$

$\Rightarrow \sigma_1 = \sigma_m + R = \sigma_{xx} + \sigma_{yy} \text{ --- (1)}$

$$\left. \begin{aligned} \epsilon_{xx} &= \frac{1}{E} (\sigma_{xx} - \nu \sigma_{yy}) \\ \epsilon_{yy} &= \frac{1}{E} (\sigma_{yy} - \nu \sigma_{xx}) \end{aligned} \right\} \Rightarrow \epsilon_{xx} + \epsilon_{yy} = \frac{1}{E} (1-\nu) (\sigma_{xx} + \sigma_{yy}) \text{ --- (2)}$$

Combining (1) & (2)  $\rightarrow$   $\boxed{\sigma_1 = \frac{E}{1-\nu} (\epsilon_{xx} + \epsilon_{yy})}$

\* ALTERNATIVELY, this problem can be solved using Mohr's circle for strain:



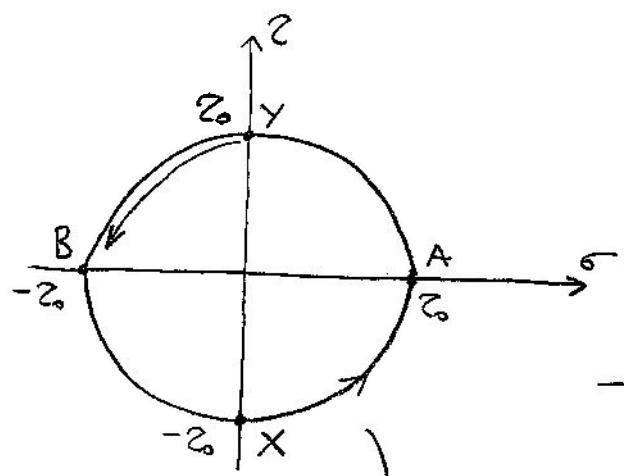
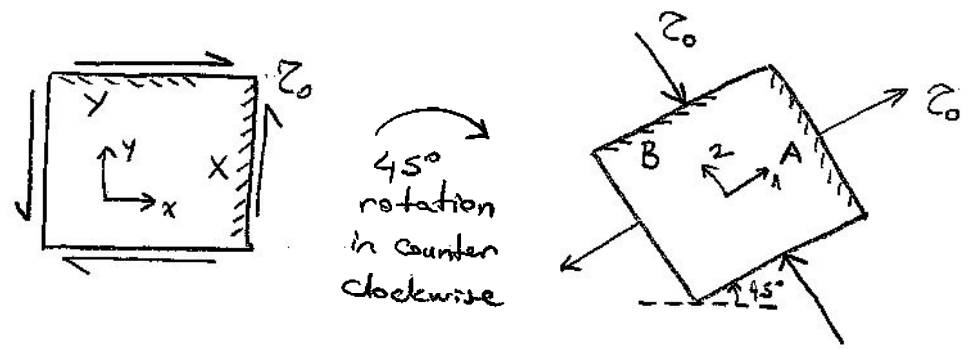
$\epsilon_m = \frac{\epsilon_{xx} + \epsilon_{yy}}{2} = \frac{\epsilon_1 - \nu \epsilon_1}{2}$

$\Rightarrow \epsilon_1 = \frac{\epsilon_{xx} + \epsilon_{yy}}{1-\nu}$

$\sigma_1 = E \cdot \epsilon_1$  for uniaxial loading

$\Rightarrow \boxed{\sigma_1 = \frac{E}{1-\nu} (\epsilon_{xx} + \epsilon_{yy})}$

Let's consider a pure shear loading and 45° transformation:



Mohr's circle corresponding to a pure shear loading is given in the sketch. After 45° C-Cwise rotation surface X transforms to A, and Y to B.

- We can write in the transformed state:

$$\left. \begin{aligned} \epsilon_1 &= \frac{1}{E} (\tau_0 + \nu \tau_0) \\ \epsilon_2 &= \frac{1}{E} (-\tau_0 - \nu \tau_0) \end{aligned} \right\} \Rightarrow \epsilon_1 = \frac{(1+\nu)}{E} \tau_0 = -\epsilon_2$$

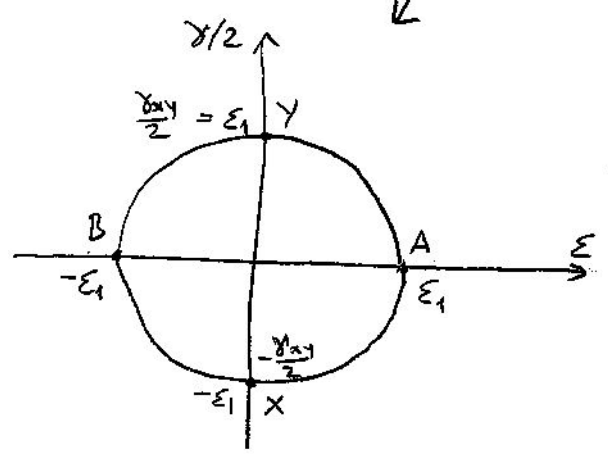
- On the other hand, in original state:

$$\gamma_{xy} = \frac{1}{\mu} \tau_0$$

it's apparent from strain Mohr's circle that  $\frac{\gamma_{xy}}{2} = \epsilon_1$

$$\Rightarrow 2 \epsilon_1 = \frac{1}{\mu} \tau_0$$

$$\frac{2(1+\nu)}{E} \tau_0 = \frac{1}{\mu} \tau_0 \Rightarrow \boxed{\mu = \frac{E}{2(1+\nu)}}$$



Problem 3 For an isotropic linear elastic solid

(3)

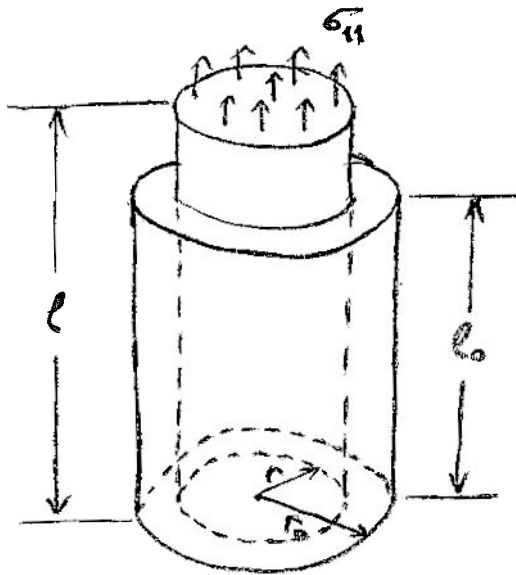
$$\sigma_{ij} = \lambda \varepsilon_{kk} \delta_{ij} + 2\mu \varepsilon_{ij}$$

$$\begin{aligned} a) \quad \sigma_{11} &= \lambda (\varepsilon_{11} + \varepsilon_{22} + \varepsilon_{33}) + 2\mu \varepsilon_{11} \\ \sigma_{22} &= \lambda (\varepsilon_{11} + \varepsilon_{22} + \varepsilon_{33}) + 2\mu \varepsilon_{22} \end{aligned} \quad \left. \begin{aligned} \sigma_{12} &= 2\mu \varepsilon_{12} \\ \sigma_{13} &= 2\mu \varepsilon_{13} \end{aligned} \right\}$$

noting that the strain ( $\varepsilon_{ij}$ ) is a unitless quantity, the Lamé's constants ( $\lambda, \mu$ ) have the unit of stress [Force/Area]  $\rightarrow$  MPa, msi, etc.

b) For the uniaxial stress state, we have

$$\left. \begin{aligned} \varepsilon_{11} &= \frac{1}{E} \sigma_{11} \\ \varepsilon_{22} &= \frac{1}{E} (-\nu \sigma_{11}) = -\nu \varepsilon_{11} \\ \varepsilon_{33} &= \frac{1}{E} (-\nu \sigma_{11}) = -\nu \varepsilon_{11} \end{aligned} \right\} \begin{aligned} &\text{the unit of } E \text{ is } \frac{N}{m^2} = Pa, \\ &\nu \text{ is unitless (non-dimensional)} \end{aligned}$$



If the undeformed radius and length are  $r_0$  and  $l_0$ , and the radius and length after deformation are  $r$  and  $l$ , then

$$\varepsilon_{11} = \frac{l - l_0}{l_0} \quad \varepsilon_{22} = \frac{r - r_0}{r_0}$$

$$\nu = -\frac{\varepsilon_{22}}{\varepsilon_{11}} = \frac{r_0 - r}{r_0} \frac{l_0}{l - l_0}$$

Problem 4 For an isotropic linear elastic solid

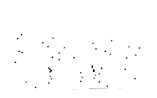
$$\sigma_{ij} = \lambda \epsilon_{kk} \delta_{ij} + 2\mu \epsilon_{ij} \quad \text{--- (1)}$$

$$\epsilon_{kk} = \epsilon_{kl} \delta_{kl} \quad \text{--- (2)}$$

$$\begin{aligned} \epsilon_{ij} &= \frac{1}{2} (\epsilon_{ij} + \epsilon_{ji}) \quad \leftarrow \text{to preserve the symmetry of } \epsilon_{ij} \\ &= \frac{1}{2} (\epsilon_{kl} \delta_{ik} \delta_{jl} + \epsilon_{kl} \delta_{jk} \delta_{il}) \quad \text{--- (3)} \end{aligned}$$

substituting (2) & (3) into (1), we get

$$\sigma_{ij} = \underbrace{\left[ \lambda \delta_{ij} \delta_{kl} + \mu (\delta_{ik} \delta_{jl} + \delta_{il} \delta_{jk}) \right]}_{C_{ijkl}} \epsilon_{kl}$$



Problem 5

$$S_{ij} = \sigma_{ij} - \frac{1}{3} \sigma_{kk} \delta_{ij} \text{ ----- (1)}$$

$$e_{ij} = \varepsilon_{ij} - \frac{1}{3} \varepsilon_{kk} \delta_{ij} \text{ ----- (2)}$$

from constitutive relation

$$\sigma_{ij} = \lambda \varepsilon_{kk} \delta_{ij} + 2\mu \varepsilon_{ij} \text{ ----- (3)}$$

$$\Rightarrow \sigma_{kk} = \lambda \varepsilon_{kk} \cdot 3 + 2\mu \varepsilon_{kk}$$

$$\Rightarrow \underbrace{\frac{\sigma_{kk}}{3}}_p = \underbrace{\left(\lambda + \frac{2}{3}\mu\right)}_K \varepsilon_{kk} \text{ ----- (4)}$$

K: bulk modulus

substituting (3) & (4) in (1) gives:

$$\begin{aligned} S_{ij} &= \lambda \varepsilon_{kk} \delta_{ij} + 2\mu \varepsilon_{ij} - \left(\lambda + \frac{2}{3}\mu\right) \varepsilon_{kk} \delta_{ij} \\ &= 2\mu \underbrace{\left(\varepsilon_{ij} - \frac{1}{3} \varepsilon_{kk} \delta_{ij}\right)}_{e_{ij}} \Rightarrow \boxed{S_{ij} = 2\mu e_{ij}} \end{aligned}$$

$$\text{And from (4)} \longrightarrow \boxed{p = -K \varepsilon_{kk}}$$

This analysis shows that any stress state can be decomposed into hydrostatic ( $p$ ) and deviatoric ( $S_{ij}$ ) parts, and hydrostatic part is responsible for volume change ( $\varepsilon_{kk}$ ) in the body while the deviatoric part generates shape change ( $e_{ij}$ ) without affecting on volume.